You are graded on your work, with partial credit where it is deserved.

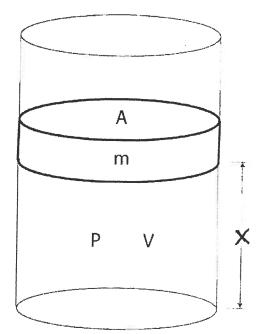
Please give clear, well-organized solutions, and please show enough work to be convincing.

1. We will consider the vibrations of a cylinder with cross-sectional area A and mass m, floating above an ideal gas with heat capacity  $C_v$  at constant volume and  $C_P$  at constant pressure, and with  $\gamma \equiv C_P / C_V$ . No energy leaves or enters the gas, so the vibrations are adiabatic.

You may assume the ideal gas equation of state PV = NkT, and the fact that E = E(T) for an ideal gas.

But please do not assume other properties, such as the relation between V and P or T during an adiabatic process, since these are equivalent to some of what is asked for below.

(a) (4) Write down the first law of thermodynamics, relating the change in internal energy dE to the heat dQ added to the gas and the work dW done by the gas\*. dE = dQ - dW



(b) (4) Using the first law, obtain the relation between  $C_v$ , dE, and dT.

$$C_{V} = \frac{dQ}{dT} \text{ with } V \text{ fixed, so that } \frac{dW}{dV} = P dV = 0$$

$$= \frac{dE}{dT} \implies dE = C_{V} dT$$

$$[or C_{V} = (\frac{\partial E}{\partial T})_{V}]$$

(c) (4) Using the first law, plus the ideal gas equation of state, show that  $C_P - C_V = \text{constant}$ , and obtain the constant in terms of the number of particles N and the Boltzmann constant k.

$$C_{p} = \frac{dQ}{dT} \text{ with } P \text{ fixed, so that } dW = P dV = d(PV)$$

$$= \frac{d(E + PV)}{dT}$$

$$= C_{V} + \frac{\partial(NkT)}{\partial T} \text{ since } PV = NkT,$$

$$= C_{V} + Nk$$

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$$\Rightarrow C_{P} - C_{V} = Nk$$

<sup>\*</sup> This is a slight change of notation, and with no explicit indication that Q and W are not proper variables.

(d) (4) For an adiabatic process, show that

$$C_{V}dT = -PdV.$$
adjubitic means  $dQ = 0$ , so  $dE = dQ - dW = -dW$ 
or  $C_{V}dT = -PdV$ 

(e) (4) Using the above equation, and the ideal gas equation of state, show that (for an adiabatic process)

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$$\frac{C_V}{C_P - C_V} (PdV + VdP) = -PdV.$$

$$\frac{C_V}{Nk} \Rightarrow C_V dT = \frac{C_V}{Nk} d(PV)$$

$$\frac{C_V}{C_P - C_V} (PdV + VdP) = -PdV$$

$$from (c) \qquad \qquad from (d)$$
(f) (4) Now use the above result to obtain a simple relation between  $\frac{dP}{dV}$  and  $\frac{P}{V}$ .
$$\frac{C_V}{C_P - C_V} = \frac{1}{V - 1} \Rightarrow PdV + VdP = -(V - 1) PdV$$

$$\Rightarrow VdP = -VPdV$$

$$\Rightarrow \sqrt{dP} = -VPdV$$

(g) (4) If F is the upward force on the cylinder due to the gas, and  $X = X_0 + x$  is the length of the cylindrical volume of gas (as shown on the first page), show that for small amplitude oscillations

$$\frac{dF}{dX} \approx -\gamma \frac{F_0}{X_0}$$
, where  $F_0 = mg + P_{atm} A$ 

and  $X_0$  is the value of X in equilibrium.

$$F = PA \text{ and } V = XA, \text{ so from above}$$

$$\frac{dF}{dX} = -\Im \frac{F}{X} \approx -\Im \frac{F_0}{X_0}$$
where in equilibrium  $F_0 = mg + P_{2tm}A$ 

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(h) (4) Now show that the adiabatic vibrations of the cylinder are described by

$$\frac{d^2x}{dt^2} = -Kx$$

while also obtaining K in terms of m,  $F_0$ , and the other constants in Part (g).

[Hint: first note that the net force  $\Delta F$  on the cylinder is given by  $\Delta F / x \approx dF / dX$ .]

$$\frac{\Delta F}{\chi} \approx \frac{dF}{d\chi} \approx -\gamma \frac{F_0}{\chi_0} \quad \text{and} \quad \Delta F = m \frac{d^2 \chi}{dt^2}$$

$$\therefore m \frac{d^2 \chi}{dt^2} = -\gamma \frac{F_0}{\chi_0} \chi \Rightarrow \boxed{\frac{d^2 \chi}{dt^2} = -\kappa \chi}, \quad \boxed{\kappa = \frac{\gamma F_0}{m \chi_0}}$$

(i) (4) Finally, obtain  $\gamma \equiv C_P / C_V$  in terms of the angular frequency  $\omega = 2\pi / T$  with which the cylinder vibrates. [Here T is the period of oscillation.]

s the period of oscillation.]
$$\chi = \chi_{0} \cos(\omega t + \phi) \Rightarrow \frac{d^{2}\chi}{dt^{2}} = -\omega^{2}\chi$$

$$\Rightarrow \omega^{2} = \chi = \frac{\nabla F_{0}}{m \chi_{0}}$$

$$\Rightarrow \sqrt{\frac{m \chi_{0}}{F_{0}}} \omega^{2}$$

(j) (4) How would the equation of Part (f) above, and the final result for the vibrational angular frequency  $\omega$ , change if the gas were held at constant temperature T, so that the vibrations were isothermal rather than adiabatic?

$$PV = NkT = constant$$

$$\Rightarrow PdV + VdP = 0$$

$$\Rightarrow \frac{dP}{dV} = -\frac{P}{V}$$
and 
$$\omega^2 = \frac{F_0}{mX_0}$$
 [with  $V$  replaced by 1]

- 2. For a (classical, monatomic, nonrelativistic) ideal gas we had  $Z = \frac{1}{N!} \left( \frac{V}{\lambda_{th}^3} \right)^N$ .
- (a) (5) By replacing a sum by an integral, obtain Stirling's approximation  $\ln N! \approx N \ln N N$ .

$$\frac{\int \ln N!}{\ln n} = \ln \frac{n}{n} = \frac{1}{n} \ln n \quad \text{an} \quad$$

(b) (5) Then obtain a simple expression for  $\ln Z \text{ in terms of } N, \lambda_{th} = \frac{h}{(2\pi mkT)^{1/2}}, \text{ and } v = \frac{V}{N}.$   $V = \frac{V}{N}.$  V

(c) (5) Using the fact that 
$$\langle E \rangle = \sum_{j} p_{j} E_{j}$$
, where  $p_{j} = \frac{e^{-E_{j}/kT}}{Z}$ , obtain  $\langle E \rangle$  in terms of  $\frac{\partial \ln Z}{\partial T}$ .

$$\frac{\partial \ln Z}{\partial T} = \frac{1}{Z} \frac{\partial}{\partial T} \sum_{j} e^{-E_{j}/kT} = \frac{1}{Z} \sum_{j} e^{-E_{j}/kT} \left(-\frac{E_{j}}{k}\right) \left(-\frac{1}{T^{2}}\right)$$

$$= \frac{1}{kT^{2}} \sum_{j} p_{j} E_{j}$$

$$= \langle E \rangle$$

$$\Rightarrow \langle E \rangle = kT^{2} \frac{\partial \ln Z}{\partial T}$$

(d) (5) Use your result above to calculate  $\langle E \rangle$  for this ideal gas.

$$|\overline{\langle E \rangle}| = |RT^2 \frac{3}{3T} \left[ N \ln T^{3/2} + \text{function of } N \text{ and } V \right]$$

$$= |RT^2 \cdot N \cdot \frac{3}{2} \frac{1}{T}$$

$$= \left[ \frac{3}{2} N k T \right]$$

- 3. Recall that the Carnot cycle for a Carnot engine consists of two isothermal processes, at temperatures  $T_{hot}$ and  $T_{cold}$  (with heat  $Q_{hot}$  received form one reservoir, and heat  $Q_{cold}$  given to the other reservoir) and two adiabatic processes.
- (a) (5) For any heat engine doing work W, what is the definition of the efficiency?

$$e = \frac{W}{Q_{hot}}$$

(b) (5) For a Carnot engine, relate  $Q_{cold}/Q_{hot}$  to  $T_{cold}/T_{hot}$ . [Hint: Consider  $\Delta S$  for one cycle.]

$$\Delta S = + \frac{Q_{hot}}{T_{hot}} - \frac{Q_{cold}}{T_{cold}} \quad \text{for Carnot cycle}$$

$$\frac{Q_{cold}}{Q_{hot}} = \frac{T_{cold}}{T_{hot}} \quad \text{for Carnot}$$

$$\frac{Q_{cold}}{Q_{hot}} = \frac{T_{cold}}{T_{hot}} \quad \text{for Carnot}$$

$$\frac{Q_{hot}}{T_{hot}} = \frac{Q_{hot} - Q_{cold}}{Q_{hot}} = 1 - \frac{Q_{cold}}{Q_{hot}} = 1 - \frac{T_{cold}}{T_{hot}}$$

(c) (5) Obtain the efficiency for a Carnot engine in terms of  $T_{cold}$  and  $T_{hot}$ .

(d) (5) If the operating temperatures are 900 K and 300 K, what is the Carnot efficiency?

4. (20) Two systems, 1 and 2, are separated by a partition that permits energy and particles to flow through, and which is also movable. Then we have

$$E = E_1 + E_2$$
 ,  $V = V_1 + V_2$  ,  $N = N_1 + N_2$  ,  $S = S_1 + S_2$ 

where the total energy E, total volume V, and total number of particles N are constant, but  $E_1$ ,  $V_1$ , and  $N_1$  can vary. Recall that

$$dS_{i} = \frac{1}{T_{i}} dE_{i} - \frac{P_{i}}{T_{i}} dV_{i} + \frac{\mu_{i}}{T_{i}} dN_{i} \qquad (i = 1, 2)$$

where T is the temperature, P the pressure, and  $\mu$  the chemical potential.

1 2

$$E_2 = E - E_1$$

$$V_2 = V - V_1$$

$$N_2 = N - N_1$$

By maximizing the total entropy, determine the 3 conditions for the two systems to be in thermal, mechanical, and chemical equilibrium.

$$0 = \frac{35}{3E_1} = \frac{35_1}{3E_1} + \frac{35_2}{3E_2} \left(\frac{3E_2}{3E_1}\right) = \frac{1}{T_1} - \frac{1}{T_2} \Rightarrow \boxed{T_2 = T_1}$$

$$0 = \frac{35}{V_1} = \frac{35_1}{3V_1} + \frac{35_2}{3V_2} \left(\frac{3V_2}{3V_1}\right) = -\frac{P_1}{T_1} + \frac{P_2}{T_2} \Rightarrow \boxed{P_2 = P_1}$$

$$0 = \frac{35}{3N_1} = \frac{35_1}{3N_1} + \frac{35_2}{3N_2} \frac{3N_2}{3N_1} = \frac{\mathcal{U}_1}{T_1} - \frac{\mathcal{U}_2}{T_2} \Rightarrow \mathcal{U}_2 = \mathcal{U}_1$$